

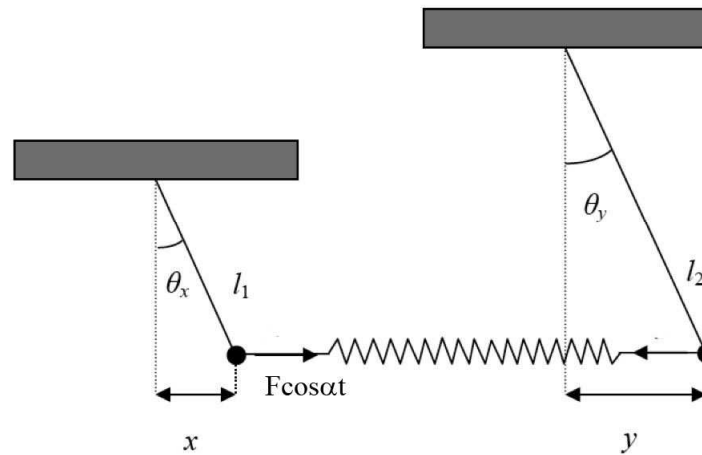
Lecture 4

Normal Modes

◇ Coupled driven oscillators

◇ Double pendulum

The damped driven pendulum



$$m_1 \ddot{x} = -\gamma \dot{x} - m_1 g x / l_1 + k(y - x) + F \cos \alpha t$$

$$m_2 \ddot{y} = -\gamma \dot{y} - m_2 g y / l_2 - k(y - x)$$

$$\begin{pmatrix} \frac{d^2}{dt^2} + \frac{\gamma}{m_1} \frac{d}{dt} + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & \frac{d^2}{dt^2} + \frac{\gamma}{m_2} \frac{d}{dt} + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix} = \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} \text{Re}(e^{i\alpha t})$$

$$\begin{pmatrix} \frac{d^2}{dt^2} + \frac{\gamma}{m_1} \frac{d}{dt} + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & \frac{d^2}{dt^2} + \frac{\gamma}{m_2} \frac{d}{dt} + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix} = \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} \text{Re}(e^{i\omega t})$$

CF

$$\begin{pmatrix} x \\ y \end{pmatrix} = \text{Re} \left(\begin{pmatrix} X \\ Y \end{pmatrix} e^{i\omega t} \right)$$

$$\begin{pmatrix} -\omega^2 + i \frac{\gamma}{m_1} \omega + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & -\omega^2 + i \frac{\gamma}{m_2} \omega + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix} \begin{pmatrix} X \\ Y \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix} \quad (1)$$

$$\begin{vmatrix} -\omega^2 + i \frac{\gamma}{m_1} \omega + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & -\omega^2 + i \frac{\gamma}{m_2} \omega + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{vmatrix} = 0$$

Eigenvalue eq.

Substitute (complex) eigenvalues in (1) to obtain eigenvectors

Simple case

$$m_1 = m_2 = m \quad l_1 = l_2 = l$$

$$\begin{vmatrix} -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{k}{m}\right) & -\frac{k}{m} \\ -\frac{k}{m} & -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{k}{m}\right) \end{vmatrix} = 0$$

$$-\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l}\right) = 0 \quad \text{or} \quad -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{2k}{m}\right) = 0$$

$$\bar{\omega}_{1,2} = i\frac{\gamma}{2m} \pm \sqrt{\omega_{1,2}^2 - \left(\frac{\gamma}{2m}\right)^2} \quad \text{Eigenvalues}$$

$$\omega_1^2 = \frac{g}{l} \quad \text{or} \quad \omega_2^2 = \frac{g}{l} + 2\frac{k}{m}$$

$\gamma = 0$ eigenvalues (c.f. previous result)

Eigenvectors

$$\begin{pmatrix} -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{k}{m}\right) & -\frac{k}{m} \\ -\frac{k}{m} & -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{k}{m}\right) \end{pmatrix} \begin{pmatrix} X \\ Y \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$

$$-\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l}\right) = 0 \quad \text{or} \quad -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{2k}{m}\right) = 0$$

$$\bar{\omega}_{1,2} = i\frac{\gamma}{2m} \pm \sqrt{\omega_{1,2}^2 - \left(\frac{\gamma}{2m}\right)^2} \quad \text{Eigenvalues}$$

$$\underline{\omega = \bar{\omega}_1} \quad \begin{pmatrix} \frac{k}{m} & -\frac{k}{m} \\ -\frac{k}{m} & \frac{k}{m} \end{pmatrix} \begin{pmatrix} X_1 \\ Y_1 \end{pmatrix} = 0$$

$$\begin{pmatrix} X_1 \\ Y_1 \end{pmatrix} = A_1 e^{i\phi_1} \begin{pmatrix} 1 \\ 1 \end{pmatrix}$$

$$\underline{\omega = \bar{\omega}_2} \quad \begin{pmatrix} -\frac{k}{m} & -\frac{k}{m} \\ \frac{k}{m} & \frac{k}{m} \end{pmatrix} \begin{pmatrix} X_2 \\ Y_2 \end{pmatrix} = 0$$

$$\begin{pmatrix} X_2 \\ Y_2 \end{pmatrix} = A_2 e^{i\phi_2} \begin{pmatrix} 1 \\ 1 \end{pmatrix} \quad \text{Eigenvectors}$$

Eigenvectors

$$\begin{pmatrix} -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{k}{m}\right) & -\frac{k}{m} \\ -\frac{k}{m} & -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{k}{m}\right) \end{pmatrix} \begin{pmatrix} X \\ Y \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$

$$-\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l}\right) = 0 \quad \text{or} \quad -\omega^2 - i\frac{\gamma}{m}\omega + \left(\frac{g}{l} + \frac{2k}{m}\right) = 0$$

$$\bar{\omega}_{1,2} = i\frac{\gamma}{2m} \pm \sqrt{\omega_{1,2}^2 - \left(\frac{\gamma}{2m}\right)^2} \quad \text{Eigenvalues}$$

$$\begin{aligned} \begin{pmatrix} x \\ y \end{pmatrix} &= \text{Re} \left(A_1 e^{i\phi_1} \begin{pmatrix} 1 \\ 1 \end{pmatrix} e^{-\frac{\gamma}{2m}t} e^{it\sqrt{\omega_1^2 - \left(\frac{\gamma}{2m}\right)^2}} \right) + \text{Re} \left(A_2 e^{i\phi_2} \begin{pmatrix} 1 \\ -1 \end{pmatrix} e^{-\frac{\gamma}{2m}t} e^{it\sqrt{\omega_2^2 - \left(\frac{\gamma}{2m}\right)^2}} \right) \\ &= e^{-\frac{\gamma}{2m}t} \left(A_1 \begin{pmatrix} 1 \\ 1 \end{pmatrix} \cos \left(t\sqrt{\omega_1^2 - \left(\frac{\gamma}{2m}\right)^2} + \phi_1 \right) + A_2 \begin{pmatrix} 1 \\ -1 \end{pmatrix} \cos \left(t\sqrt{\omega_2^2 - \left(\frac{\gamma}{2m}\right)^2} + \phi_2 \right) \right) \end{aligned}$$

Decoupling method

$$m\ddot{x} = -mg\frac{x}{l} + k(y - x) - \gamma\dot{x}$$

$$m\ddot{y} = -mg\frac{y}{l} - k(y - x) - \gamma\dot{y}$$

$$m(\ddot{x} + \ddot{y}) = -m\frac{g}{l}(x + y) - \gamma(\dot{x} + \dot{y})$$

$$\ddot{q}_1 + \frac{\gamma}{m}\dot{q}_1 + \omega_1^2 q_1 = 0$$

$$m(\ddot{x} - \ddot{y}) = (-m\frac{g}{l} - 2k)(x - y) - \gamma(\dot{x} - \dot{y})$$

$$\ddot{q}_2 + \frac{\gamma}{m}\dot{q}_2 + \omega_2^2 q_2 = 0$$

Two 2nd order differential equations - techniques discussed last term give

$$q_1 = A_1 \cos(\omega_1' t + \phi_1) e^{-\alpha t}$$

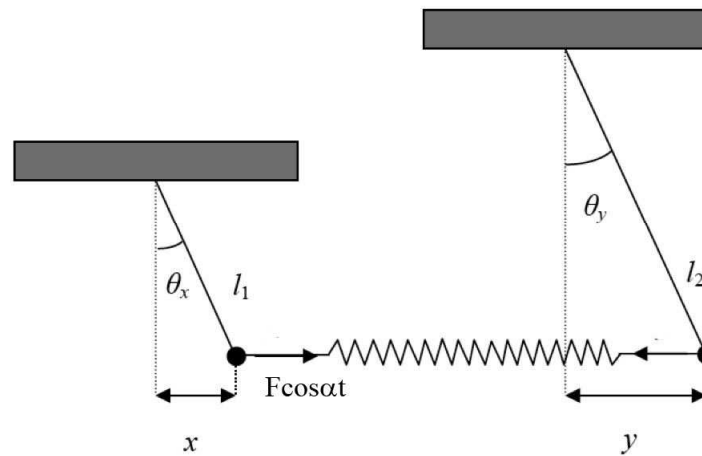
$$\omega_1' = \sqrt{\omega_1^2 - \alpha^2}$$

$$q_2 = A_2 \cos(\omega_2' t + \phi_2) e^{-\alpha t}$$

$$\omega_2' = \sqrt{\omega_2^2 - \alpha^2}$$

$$\alpha = \frac{\gamma}{2m}$$

The damped driven pendulum - the Particular Integral



$$m_1 \ddot{x} = -\gamma \dot{x} - m_1 g x / l_1 + k(y - x) + F \cos \alpha t$$

$$m_2 \ddot{y} = -\gamma \dot{y} - m_2 g y / l_2 - k(y - x)$$

$$\begin{pmatrix} \frac{d^2}{dt^2} + \frac{\gamma}{m_1} \frac{d}{dt} + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & \frac{d^2}{dt^2} + \frac{\gamma}{m_2} \frac{d}{dt} + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix} = \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} \text{Re}(e^{i\alpha t})$$

$$\begin{pmatrix} \frac{d^2}{dt^2} + \frac{\gamma}{m_1} \frac{d}{dt} + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & \frac{d^2}{dt^2} + \frac{\gamma}{m_2} \frac{d}{dt} + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix} = \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} \text{Re}(e^{i\alpha t})$$

PI

$$\begin{pmatrix} x \\ y \end{pmatrix} = \text{Re} \left(\begin{pmatrix} P \\ Q \end{pmatrix} e^{i\alpha t} \right)$$

$$\begin{pmatrix} -\alpha^2 + i \frac{\gamma}{m_1} \alpha + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & -\alpha^2 + i \frac{\gamma}{m_2} \alpha + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix} \begin{pmatrix} P \\ Q \end{pmatrix} \equiv \mathbf{M} \mathbf{P} = \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix}$$

$$\mathbf{P} \equiv \begin{pmatrix} P \\ Q \end{pmatrix} = \mathbf{M}^{-1} \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} \qquad \mathbf{M} = \begin{pmatrix} -\alpha^2 + i \frac{\gamma}{m_1} \alpha + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & -\alpha^2 + i \frac{\gamma}{m_2} \alpha + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix}$$

$$\begin{pmatrix} x \\ y \end{pmatrix} = \text{Re} \left(\mathbf{M}^{-1} \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} e^{i\alpha t} \right)$$

$$\begin{pmatrix} x \\ y \end{pmatrix} = \text{Re} \left(\mathbf{M}^{-1} \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} e^{i\alpha t} \right)$$

$$\mathbf{M} = \begin{pmatrix} -\alpha^2 + i \frac{\gamma}{m_1} \alpha + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & -\alpha^2 + i \frac{\gamma}{m_2} \alpha + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix}$$

e.g. $m_1 = m_2 = m \quad l_1 = l_2 = l \quad \gamma = 0 \quad \left(\omega_1^2 = \frac{g}{l}, \quad \omega_2^2 = \frac{g}{l} + 2 \frac{k}{m} \right)$

$$\mathbf{M} = \begin{pmatrix} -\alpha^2 + \left(\frac{g}{l} + \frac{k}{m} \right) & -\frac{k}{m} \\ -\frac{k}{m} & -\alpha^2 + \left(\frac{g}{l} + \frac{k}{m} \right) \end{pmatrix}$$

$$\mathbf{M}^{-1} = \frac{1}{\text{Det}M} \begin{pmatrix} -\alpha^2 + \left(\frac{g}{l} + \frac{k}{m} \right) & \frac{k}{m} \\ \frac{k}{m} & -\alpha^2 + \left(\frac{g}{l} + \frac{k}{m} \right) \end{pmatrix}$$

$$\text{Det}M = \left(-\alpha^2 + \left(\frac{g}{l} + \frac{k}{m} \right) \right)^2 - \left(\frac{k}{m} \right)^2 = (\alpha^2 - \omega_1^2)(\alpha^2 - \omega_2^2)$$

$$\begin{aligned} \begin{pmatrix} x \\ y \end{pmatrix} &= \left(\frac{F}{m} \cos \alpha t \right) \frac{1}{(\alpha^2 - \omega_1^2)(\alpha^2 - \omega_2^2)} \begin{pmatrix} -\alpha^2 + \left(\frac{g}{l} + \frac{k}{m} \right) \\ \frac{k}{m} \end{pmatrix} = \left(\frac{F}{m} \cos \alpha t \right) \frac{1}{(\alpha^2 - \omega_1^2)(\alpha^2 - \omega_2^2)} \begin{pmatrix} -\alpha^2 + (\omega_2^2 + \omega_1^2)/2 \\ (\omega_2^2 - \omega_1^2)/2 \end{pmatrix} \\ &= \left(-\frac{F}{2m} \cos \alpha t \right) \begin{pmatrix} \frac{1}{(\alpha^2 - \omega_1^2)} + \frac{1}{(\alpha^2 - \omega_2^2)} \\ \frac{1}{(\alpha^2 - \omega_1^2)} - \frac{1}{(\alpha^2 - \omega_2^2)} \end{pmatrix} \end{aligned}$$

Forced oscillations - decoupling method

$$m_1 = m_2 = m \quad l_1 = l_2 = l$$

$$\begin{aligned} m\ddot{x} + \gamma\dot{x} + m\frac{g}{l}x + k(x - y) &= F \cos \alpha t \\ m\ddot{y} + \gamma\dot{y} + m\frac{g}{l}y + k(y - x) &= 0 \end{aligned}$$

$$(\ddot{x} + \ddot{y}) + \frac{\gamma}{m}(\dot{x} + \dot{y}) + \frac{g}{l}(x + y) = \frac{F}{m} \cos \alpha t$$

$$(\ddot{x} - \ddot{y}) + \frac{\gamma}{m}(\dot{x} - \dot{y}) + \frac{g}{l}(x - y) + \frac{2k}{m}(x - y) = \frac{F}{m} \cos \alpha t$$

$$\ddot{q}_1 + \frac{\gamma}{m}\dot{q}_1 + \frac{g}{l}q_1 = \frac{F}{m} \cos \alpha t$$

$$\ddot{q}_2 + \frac{\gamma}{m}\dot{q}_2 + \left(\frac{g}{l} + \frac{2k}{m} \right) q_2 = \frac{F}{m} \cos \alpha t$$

$$q_1 = x + y$$

$$q_2 = x - y$$

CF already done, PI ...

- $$\ddot{q}_1 + \frac{\gamma}{m} \dot{q}_1 + \frac{g}{l} q_1 = \frac{F}{m} \cos \alpha t = \operatorname{Re} \left[\frac{F}{m} \exp(i \alpha t) \right], \quad q_1 = x + y$$

Pl
$$q_1 = \operatorname{Re} \left[A_1 \exp(i \alpha t) \right]$$

$$\left(-\alpha^2 + i \alpha \frac{\gamma}{m} + \frac{g}{l} \right) A_1 = F / m$$

$$A_1 = \frac{(F / m) \exp(i \phi_1)}{\left(\left(\frac{g}{l} - \alpha^2 \right)^2 + \left(\frac{\alpha \gamma}{m} \right)^2 \right)^{1/2}} = \frac{(F / m) \exp(i \phi_1)}{\left((\omega_1^2 - \alpha^2)^2 + \left(\frac{\alpha \gamma}{m} \right)^2 \right)^{1/2}}$$

where $\tan \phi_1 = \frac{-\alpha(\gamma / m)}{(\omega_1^2 - \alpha^2)}$ and $\omega_1 = \sqrt{\frac{g}{l}}$ the undamped normal mode frequency

$$q_1 = \frac{(F / m)}{\left((\omega_1^2 - \alpha^2)^2 + \left(\frac{\alpha \gamma}{m} \right)^2 \right)^{1/2}} \cos(\alpha t + \phi_1)$$

- $$\ddot{q}_2 + \frac{\gamma}{m} \dot{q}_2 + \left(\frac{g}{l} + \frac{2k}{m} \right) q_2 = \frac{F}{m} \cos \alpha t \quad q_2 = x - y$$

PI
$$q_2 = \text{Re} \left[A_2 \exp(i \alpha t) \right]$$

$$A_2 = \frac{(F/m) \exp(i \phi_2)}{\left(\left(\frac{g}{l} + \frac{2k}{m} - \alpha^2 \right)^2 + \left(\frac{\alpha \gamma}{m} \right)^2 \right)^{1/2}} = \frac{(F/m) \exp(i \phi_2)}{\left((\omega_2^2 - \alpha^2)^2 + \left(\frac{\alpha \gamma}{m} \right)^2 \right)^{1/2}}, \quad \tan \phi_2 = \frac{-\alpha (\gamma / m)}{(\omega_2^2 - \alpha^2)}$$

$q_2 = \frac{(F/m)}{\left((\omega_2^2 - \alpha^2)^2 + \left(\frac{\alpha \gamma}{m} \right)^2 \right)^{1/2}} \cos(\alpha t + \phi_2)$	$q_1 = \frac{(F/m)}{\left((\omega_1^2 - \alpha^2)^2 + \left(\frac{\alpha \gamma}{m} \right)^2 \right)^{1/2}} \cos(\alpha t + \phi_1)$
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Finally $x = \frac{1}{2}(q_1 + q_2), \quad y = \frac{1}{2}(q_1 - q_2) \quad c.f. \begin{pmatrix} x \\ y \end{pmatrix} = \left(-\frac{F}{2m} \cos \alpha t \right) \begin{pmatrix} \frac{1}{(\alpha^2 - \omega_1^2)} + \frac{1}{(\alpha^2 - \omega_2^2)} \\ \frac{1}{(\alpha^2 - \omega_1^2)} - \frac{1}{(\alpha^2 - \omega_2^2)} \end{pmatrix}, \quad \gamma = 0$

$$\begin{pmatrix} \frac{d^2}{dt^2} + \frac{\gamma}{m_1} \frac{d}{dt} + \left(\frac{g}{l_1} + \frac{k}{m_1} \right) & -\frac{k}{m_1} \\ -\frac{k}{m_2} & \frac{d^2}{dt^2} + \frac{\gamma}{m_2} \frac{d}{dt} + \left(\frac{g}{l_2} + \frac{k}{m_2} \right) \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix} = \frac{F}{m_1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} \text{Re}(e^{i\omega t})$$

The case $m_1 = m_2 = m$, $l_1 \neq l_2$, no damping, no driving force.

Eigenvalue equation

$$\begin{vmatrix} -\omega^2 + \left(\frac{g}{l_1} + \frac{k}{m} \right) & -\frac{k}{m} \\ -\frac{k}{m} & -\omega^2 + \left(\frac{g}{l_2} + \frac{k}{m} \right) \end{vmatrix} = 0$$

$$\begin{aligned} A &= g / l_1 + k / m = \beta_1^2 + k / m \\ B &= -k / m \\ C &= g / l_2 + k / m = \beta_2^2 + k / m \end{aligned}$$

$$\begin{pmatrix} x \\ y \end{pmatrix} = \text{Re} \left\{ \begin{pmatrix} x_0 \\ y_0 \end{pmatrix} e^{i\omega t} \right\}$$

$$\omega_{1,2}^2 = \frac{1}{2} \left[(\beta_1^2 + \beta_2^2) + 2k / m \pm \sqrt{(\beta_1^2 - \beta_2^2)^2 + (2k / m)^2} \right]$$

$$\frac{x_0}{y_0} = -\frac{m}{2k} \left[(\beta_1^2 - \beta_2^2) \pm \sqrt{(\beta_1^2 - \beta_2^2)^2 + (2k / m)^2} \right]$$

$$\left(\frac{y_0}{x_0} \right)_1 = -1 / \left(\frac{y_0}{x_0} \right)_2 \equiv r$$

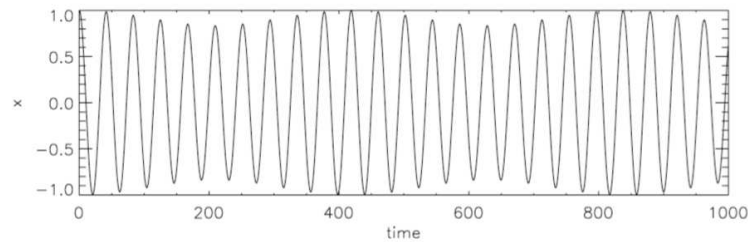
$$\left(\frac{y_0}{x_0}\right)_1 = -1 / \left(\frac{y_0}{x_0}\right)_2 \equiv r \quad (x_0)_1 = De^{i\delta_1}, \quad (x_0)_2 = De^{i\delta_2}$$

$$\mathbf{x}(t) = \begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} 1 \\ r \end{pmatrix} D \cos(\omega_1 t + \delta_1) + \begin{pmatrix} -r \\ 1 \end{pmatrix} G \cos(\omega_2 t + \delta_2)$$

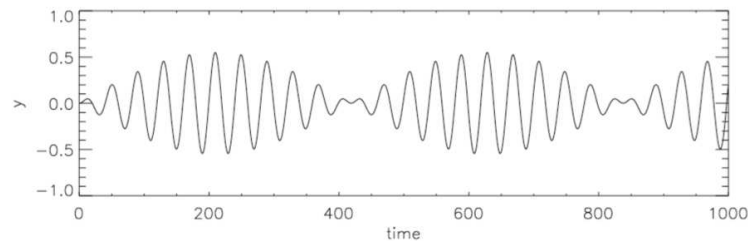
$$\text{Initial conditions : } \mathbf{x}(0) = \begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} a \\ 0 \end{pmatrix}, \quad \dot{\mathbf{x}} = 0$$

$$\bar{\omega} = \frac{\omega_1 + \omega_2}{2}, \quad \Delta\omega = \omega_1 - \omega_2$$

$$\begin{aligned} x(t) &= a [\cos \omega_1 t + r^2 \cos \omega_2 t] / (1 + r^2) \\ y(t) &= ar [\cos \omega_1 t - \cos \omega_2 t] / (1 + r^2) \end{aligned} \Rightarrow \begin{aligned} x(t) &= a \cos(\bar{\omega} t) \cos(\Delta\omega t / 2) - a \left(\frac{1 - r^2}{1 + r^2} \right) \sin(\bar{\omega} t) \sin(\Delta\omega t / 2) \\ y(t) &= 2ar \sin(\bar{\omega} t) \sin(\Delta\omega t / 2) / (1 + r^2) \end{aligned}$$



$$\left(\frac{1 - r^2}{1 + r^2} \right) a \leq |x| \leq a$$



$$0 \leq y \leq \left(\frac{2r}{1 + r^2} \right) a$$

Diagrammatic Representation of Normal Modes

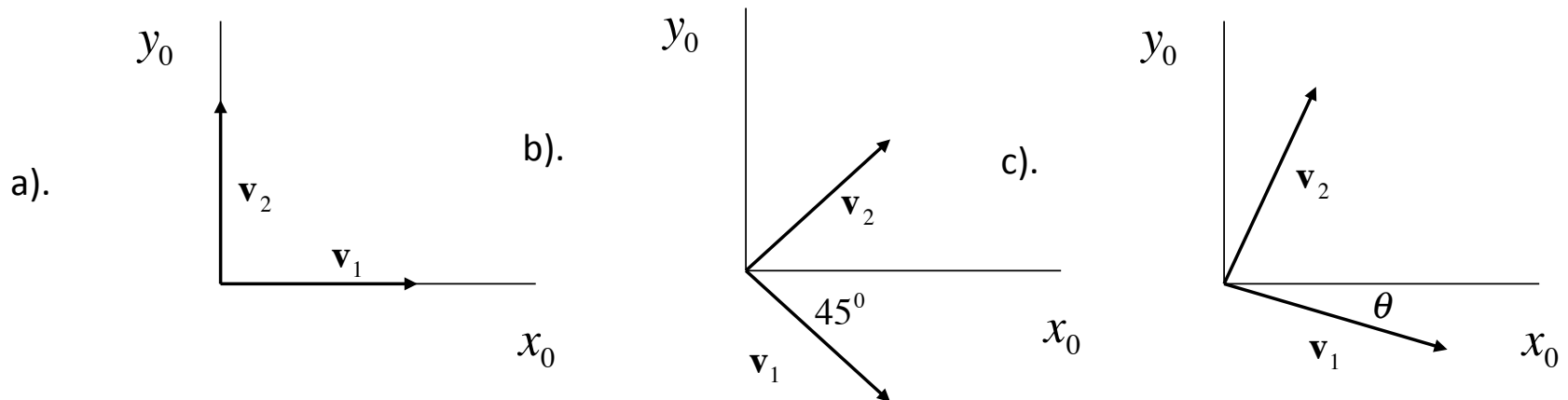
$$\mathbf{v}_{1,2} = (x_0 \mathbf{i} + y_0 \mathbf{j}) / \sqrt{x_0^2 + y_0^2}$$

$$\left(\frac{x_0}{y_0} \right)_{1,2} = -\frac{m}{2k} \left[(\beta_1^2 - \beta_2^2) \pm \sqrt{(\beta_1^2 - \beta_2^2)^2 + (2k/m)^2} \right]$$

- (a) For $k/m \rightarrow 0$ $x_0/y_0 \rightarrow -\infty$ or 0
 (b) For $k/m \rightarrow \infty$ $x_0/y_0 \rightarrow -1$ or 1

(c) Intermediate k/m

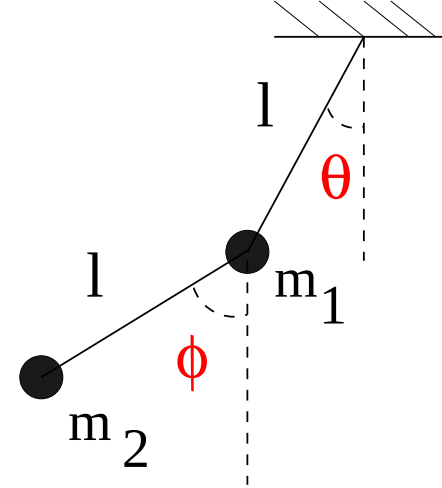
$$\frac{y_0}{x_0} = \tan \theta = \frac{-2k/m}{(\beta_1^2 - \beta_2^2) \pm \sqrt{(\beta_1^2 - \beta_2^2)^2 + (2k/m)^2}}$$



Double pendulum

small angle approximation

$$\theta, \phi \ll 1$$



$$m_1 l \ddot{\theta} = -m_1 g \theta + m_2 g (\phi - \theta)$$

$$m_2 l (\ddot{\theta} + \ddot{\phi}) = -m_2 g \phi$$

$$\frac{d^2}{dt^2} \begin{pmatrix} \theta \\ \phi \end{pmatrix} = -\frac{g}{l} \begin{pmatrix} (m_1 + m_2)/m_1 & -m_2/m_1 \\ -(m_1 + m_2)/m_1 & (m_1 + m_2)/m_1 \end{pmatrix} \begin{pmatrix} \theta \\ \phi \end{pmatrix}$$

Normal modes:

$$\begin{pmatrix} \theta \\ \phi \end{pmatrix} = \begin{pmatrix} \theta_0 \\ \phi_0 \end{pmatrix} e^{i\omega t}$$

$$\Rightarrow \begin{pmatrix} -\omega^2 + (g/l)(m_1 + m_2)/m_1 & -(g/l)(m_2/m_1) \\ -(g/l)(m_1 + m_2)/m_1 & -\omega^2 + (g/l)(m_1 + m_2)/m_1 \end{pmatrix} \begin{pmatrix} \theta_0 \\ \phi_0 \end{pmatrix} = 0$$

$$\det \begin{pmatrix} -\omega^2 + (g/l)(m_1 + m_2)/m_1 & -(g/l)(m_2/m_1) \\ -(g/l)(m_1 + m_2)/m_1 & -\omega^2 + (g/l)(m_1 + m_2)/m_1 \end{pmatrix} = 0$$

$$\Rightarrow \omega^2 - (g/l)(m_1 + m_2)/m_1 = \pm (g/l)\sqrt{m_2(m_1 + m_2)}/m_1$$

i.e.,

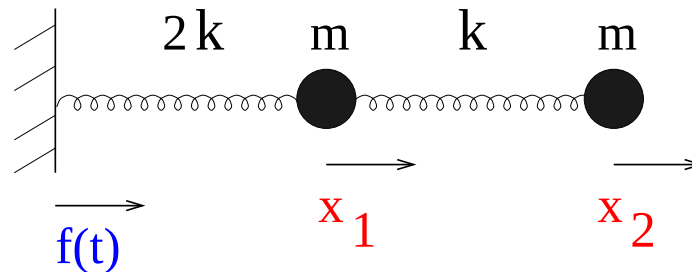
$$\omega_{\pm} = \sqrt{\frac{g}{l} \frac{1}{1 \mp \sqrt{m_2/(m_1 + m_2)}}} \quad \text{normal frequencies}$$

Example [CP4 June 2006]

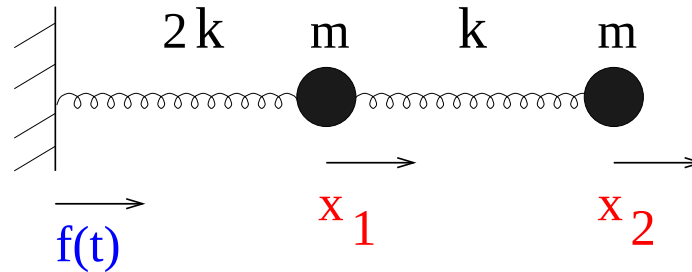
12. Two identical masses $m_1 = m_2 = m$ are connected by a massless spring with spring constant k . Mass m_1 is attached to a support by another massless spring with spring constant $2k$. The masses and springs lie along the horizontal x-axis on a smooth surface. The masses and the support are allowed to move along the x-axis only. The displacement of the support in the x-direction at time t is given by $f(t)$ and is externally controlled. Write down a system of differential equations describing the evolution of the displacements x_1 and x_2 of the masses from their equilibrium positions. [5]

Determine the frequencies of the normal modes and their amplitude ratios. [8]

The displacement of the support is given by $f(t) = A \sin(\omega t)$ with $\omega^2 = k/m$ and constant amplitude A . Find expressions for $x_1(t)$ and $x_2(t)$ assuming that any transients have been damped out by a small, otherwise negligible, damping term. [7]



◇ Coupled oscillators with a driving term:



$$m\ddot{x}_2 = -k(x_2 - x_1)$$

$$m\ddot{x}_1 = -2k(x_1 - f(t)) - k(x_1 - x_2)$$

- Homogeneous case ($f = 0$):

$$\longrightarrow \text{normal frequencies } \omega_1 = \sqrt{\frac{k}{m}(2 + \sqrt{2})} \quad , \quad \omega_2 = \sqrt{\frac{k}{m}(2 - \sqrt{2})}$$

$$\text{amplitude ratios : } (X_2/X_1)_{NM1} = 1 - \sqrt{2} \quad , \quad (X_2/X_1)_{NM2} = 1 + \sqrt{2}$$

- Driving term $f(t) = A \sin \omega t$, $\omega = \sqrt{k/m}$:

$$x_1(t) = C_1 \operatorname{Im} e^{i\omega t} \quad , \quad x_2(t) = C_2 \operatorname{Im} e^{i\omega t}$$

$$\Rightarrow \quad -\omega^2 \begin{pmatrix} C_1 \\ C_2 \end{pmatrix} = \omega^2 \begin{pmatrix} -3 & 1 \\ 1 & -1 \end{pmatrix} \begin{pmatrix} C_1 \\ C_2 \end{pmatrix} + \omega^2 \begin{pmatrix} 2 \\ 0 \end{pmatrix} A$$

$$\text{Thus : } -C_1 = -3C_1 + C_2 + 2A$$

$$-C_2 = C_1 - C_2$$

$$\Rightarrow \quad C_1 = 0, C_2 = -2A$$

$$x_1(t) = 0 \quad , \quad x_2(t) = -2A \sin \omega t$$