

We have seen

2nd-order linear ODEs with constant coefficients:

$$a_2 f'' + a_1 f' + a_0 f = h(x)$$

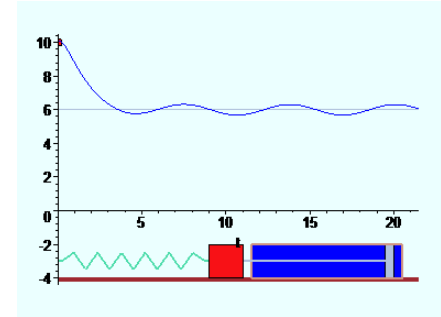
- ▷ Complementary function CF by solving *auxiliary equation*
- ▷ Particular integral PI by *trial function* with functional form of the inhomogeneous term

♠ Next: physical application to forced, damped oscillator

Oscillators

$$m\ddot{x} = - m\omega_0^2 x - m\gamma\dot{x} + mF \cos \omega t.$$

spring
friction
forcing



The associated complex equation is

$$\ddot{z} + \gamma\dot{z} + \omega_0^2 z = Fe^{i\omega t}.$$

Transients :

CF - Auxiliary equation : $z = e^{\alpha t}$.

$$\alpha^2 + \gamma\alpha + \omega_0^2 = 0 \Rightarrow \alpha = -\frac{1}{2}\gamma \pm i\sqrt{\omega_0^2 - \frac{1}{4}\gamma^2}$$

$$= -\frac{1}{2}\gamma \pm i\omega_\gamma \quad \text{where} \quad \omega_\gamma \equiv \omega_0 \sqrt{1 - \frac{1}{4}\gamma^2/\omega_0^2}.$$

$$\omega_0^2 - \frac{1}{4}\gamma^2 > 0$$

Complementary function

$$x = e^{-\gamma t/2} [A \cos(\omega_\gamma t) + B \sin(\omega_\gamma t)] = e^{-\gamma t/2} N \cos(\omega_\gamma t + \psi)$$

Constant phase "shift"

Since $\gamma > 0$, the CF $\rightarrow 0$ as $t \rightarrow \infty$ CF describes "transients"

Steady state solutions

$$\ddot{z} + \gamma\dot{z} + \omega_0^2 z = Fe^{i\omega t}.$$

No damping
exponential

Particular integral

$$x = \Re\left(\frac{Fe^{i\omega t}}{\omega_0^2 - \omega^2 + i\omega\gamma}\right).$$

i.e. the Particular integral describes the steady state solution after the transients have died away.

Since the denominator = $\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2\gamma^2} e^{i\phi}$ where $\phi \equiv \arctan\left(\frac{\omega\gamma}{\omega_0^2 - \omega^2}\right)$ the

particular integral can be written as

$$x = \frac{F\Re(e^{i(\omega t - \phi)})}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2\gamma^2}} = \frac{F \cos(\omega t - \phi)}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2\gamma^2}}$$

For $\phi > 0$, x achieves the same phase as F at t greater by $\left(\frac{\phi}{\omega}\right)$

- ϕ is called the "phase lag" of the response.

$$x = \frac{F \Re(e^{i(\omega t - \phi)})}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}} = \frac{F \cos(\omega t - \phi)}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}}$$

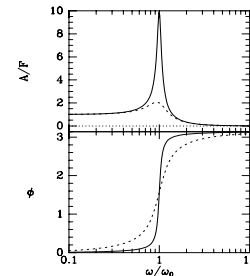
The amplitude of the response is $A = \frac{F}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}}$,

This has a maximum when $0 = \frac{dA^{-2}}{d\omega} \propto -4(\omega_0^2 - \omega^2)\omega + 2\omega\gamma^2 \Rightarrow \omega^2 = \omega_0^2 - \frac{1}{2}\gamma^2$.

$\omega_R \equiv \sqrt{\omega_0^2 - \gamma^2/2}$ is called the “resonant” frequency

The frictional coefficient causes the resonant frequency to be less than the normal frequency

$$\phi \equiv \arctan\left(\frac{\omega\gamma}{\omega_0^2 - \omega^2}\right)$$



Oscillators

$$m\ddot{x} = - m\omega_0^2 x - m\gamma\dot{x} + mF \cos \omega t.$$

spring
friction
forcing

$$\mathbf{F} = mF \cos \omega t, \quad x = \frac{F \Re(e^{i(\omega t - \phi)})}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}} = \frac{F \cos(\omega t - \phi)}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}}$$

Power Input

(steady state) $P = \frac{\partial W}{\partial t} = \mathbf{F}\dot{x}$ $W = \int_{x(t_0)}^{x(t)} F dx'$

$$\begin{aligned}
 P = \mathbf{F}\dot{x} &= mF \cos \omega t \times \frac{-F\omega \sin(\omega t - \phi)}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}} \\
 &= \frac{\omega m F^2}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}} [-\cos(\omega t) \sin(\omega t - \phi)] \\
 &= -\frac{\frac{1}{2} \omega m F^2}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}} [\sin(2\omega t - \phi) + \sin(-\phi)].
 \end{aligned}$$

Average over a period

$$\overline{P} = \frac{\frac{1}{2} \omega m F^2 \sin \phi}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 \gamma^2}}.$$

Energy dissipated

$$m\ddot{x} = - \underbrace{m\omega_0^2 x}_{\text{spring}} - \underbrace{m\gamma\dot{x}}_{\text{friction}} + \underbrace{mF \cos \omega t}_{\text{forcing}}.$$

$$\overline{D} = m\gamma \overline{\dot{x}\dot{x}} = \frac{m\gamma\omega^2 F^2 \frac{1}{2}}{(\omega_0^2 - \omega^2)^2 + \omega^2\gamma^2}.$$

$$x = \frac{F \cos(\omega t - \phi)}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2\gamma^2}}$$

$$\overline{\sin^2(\omega t - \phi)} = \frac{1}{2}$$

$$\overline{D} = \overline{P} = \frac{\frac{1}{2} \omega m F^2 \sin \phi}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2\gamma^2}} \quad \text{since} \quad \sin \phi = \gamma\omega / \sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2\gamma^2}$$

$$\left(\phi \equiv \arctan\left(\frac{\omega\gamma}{\omega_0^2 - \omega^2}\right) \right)$$

Quality Factor

$$m\ddot{x} = - \underbrace{m\omega_0^2 x}_{\text{spring}} - \underbrace{m\gamma\dot{x}}_{\text{friction}} + \underbrace{mF \cos \omega t}_{\text{forcing}}$$

Energy content of transient motion that the CF describes

$$E = \frac{1}{2}(m\dot{x}^2 + m\omega_0^2 x^2) \quad x = e^{-\gamma t/2} A \cos(\omega_\gamma t + \psi)$$

$$= \frac{1}{2} m A^2 e^{-\gamma t} \left[\frac{1}{4} \gamma^2 \cos^2 \eta + \omega_\gamma \gamma \cos \eta \sin \eta + \omega_\gamma^2 \sin^2 \eta + \omega_0^2 \cos^2 \eta \right] \quad (\eta \equiv \omega_\gamma t + \psi)$$

$$E \simeq \frac{1}{2} m (\omega_0 A)^2 e^{-\gamma t} \quad \text{small } \frac{\gamma}{\omega_0} \quad (\omega_\gamma \equiv \omega_0 \sqrt{1 - \frac{1}{4} \gamma^2 / \omega_0^2} \approx \omega_0)$$

Quality factor

$$Q \equiv \frac{E(t)}{E(t - \pi/\omega_0) - E(t + \pi/\omega_0)} \simeq \frac{1}{e^{\pi\gamma/\omega_0} - e^{-\pi\gamma/\omega_0}} = \frac{1}{2} \operatorname{csc} h(\pi\gamma/\omega_0)$$

$$\simeq \frac{\omega_0}{2\pi\gamma} \quad (\text{for small } \gamma/\omega_0).$$

Q is the inverse of the fraction of the oscillator's energy that is dissipated in one period
 - approximately the number of oscillations before the energy decays by factor e

SYSTEMS OF ORDINARY DIFFERENTIAL EQUATIONS

- more than 1 unknown function: $y_1(x), y_2(x), \dots, y_n(x)$
- set of ODEs that couple y_1, \dots, y_n

▷ physical applications: systems with more than 1 degree of freedom. dynamics couples differential equations for different variables.

Example. System of first-order differential equations:

$$y_1' = F_1(x, y_1, y_2, \dots, y_n)$$

$$y_2' = F_2(x, y_1, y_2, \dots, y_n)$$

...

$$y_n' = F_n(x, y_1, y_2, \dots, y_n)$$

An n th-order differential equation

$$y^{(n)} = G(x, y, y', y'', \dots, y^{(n-1)})$$

can be thought of as a system of n first-order equations.

- Set new variables $y_1 = y; y_2 = y'; \dots; y_n = y^{(n-1)}$
- Then the system of first-order equations

$$y_1' = y_2$$

...

$$y_{n-1}' = y_n$$

$$y_n' = G(x, y_1, y_2, \dots, y_n)$$

is equivalent to the starting n th-order equation.

- ◇ Systems of linear ODEs with constant coefficients can be solved by a generalization of the method seen for single ODE:

$$\text{General solution} = \text{PI} + \text{CF}$$

- ▷ Complementary function CF by solving *system of auxiliary equations*

- ▷ Particular integral PI from a *set of trial functions*

with functional form as the inhomogeneous terms

◇ Warm-up exercise

The variables $\psi(z)$ and $\phi(z)$ obey the simultaneous differential equations

$$3 \frac{d\phi}{dz} + 5\psi = 2z$$

$$3 \frac{d\psi}{dz} + 5\phi = 0.$$

Find the general solution for ψ .

◇ Next time we will consider explicit examples of solution of systems of ODE's with constant coefficients